

Variational Calculus (Continued)

4. Functionals w/ higher derivatives

$$J[u] = \int_0^L F(x, u, u', u'') dx$$

$$\Rightarrow \checkmark F_u - \frac{d}{dx} F_{u'} + \frac{d^2}{dx^2} F_{u''} = 0 \quad \text{Euler-Lagrange}$$

on either $x=0, L$

$$\begin{aligned} \textcircled{1} \quad u &= \bar{u} \quad \text{or} \quad F_{u'} - \frac{d}{dx} F_{u''} = 0 \\ \textcircled{2} \quad u' &= \bar{u}' \quad \text{or} \quad F_{u''} = 0 \end{aligned}$$

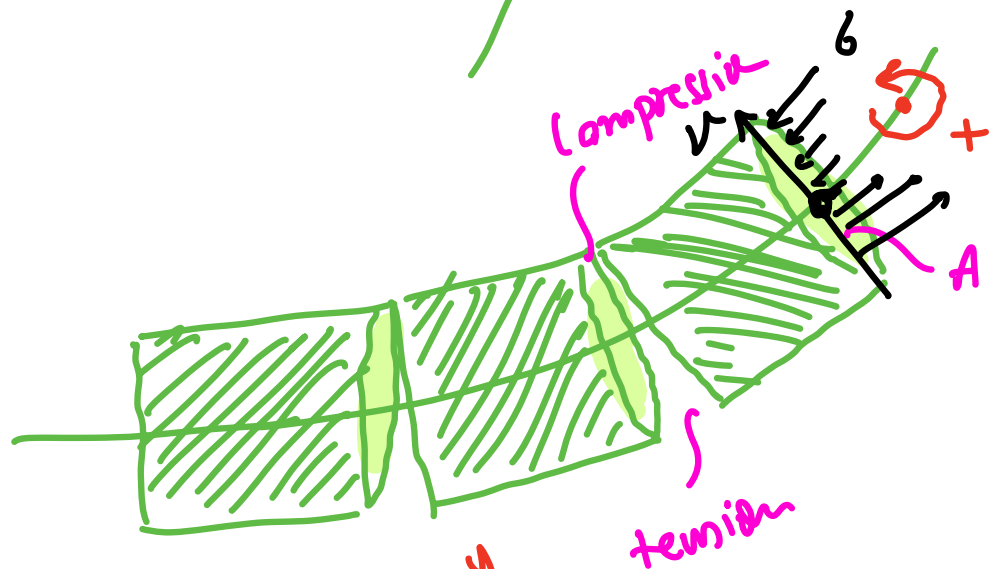
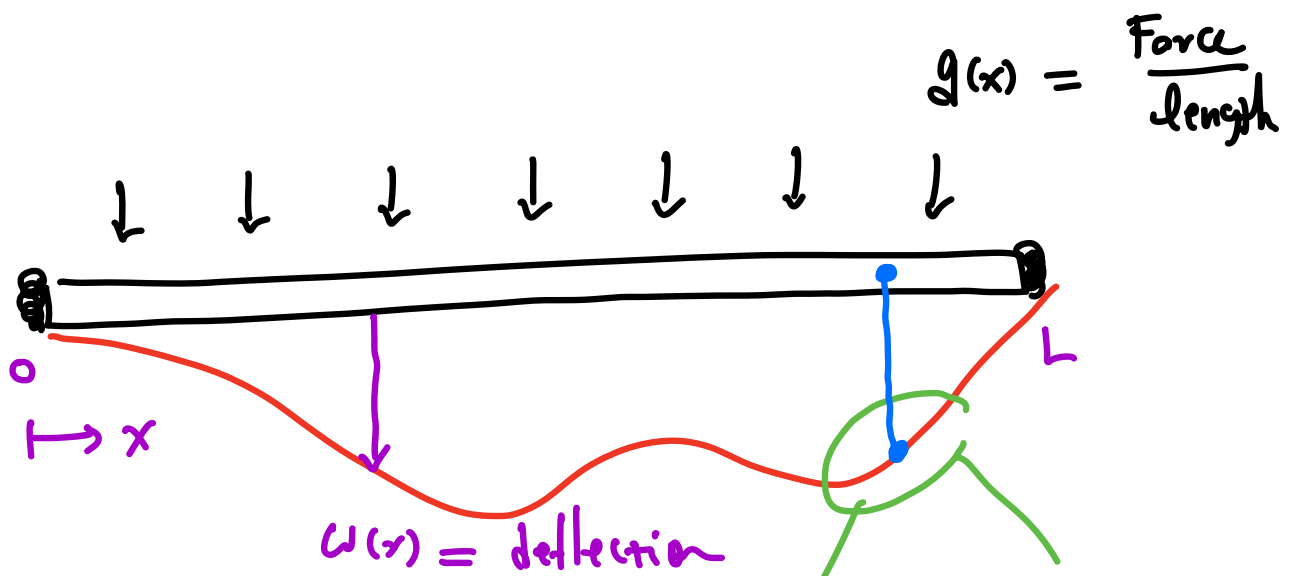
pf.

$$\begin{aligned} \delta J &= \int_0^L \left[F_u - \frac{d}{dx} F_{u'} + \frac{d^2}{dx^2} F_{u''} \right] \delta u \\ &\quad + \left(F_{u'} - \frac{d}{dx} F_{u''} \right) \delta u \Big|_0^L + F_{u''} \delta u' \Big|_0^L = 0 \end{aligned}$$

① δu to vanish on $x=0$ and L

② now pick δu such that it is free at $x=0, L$

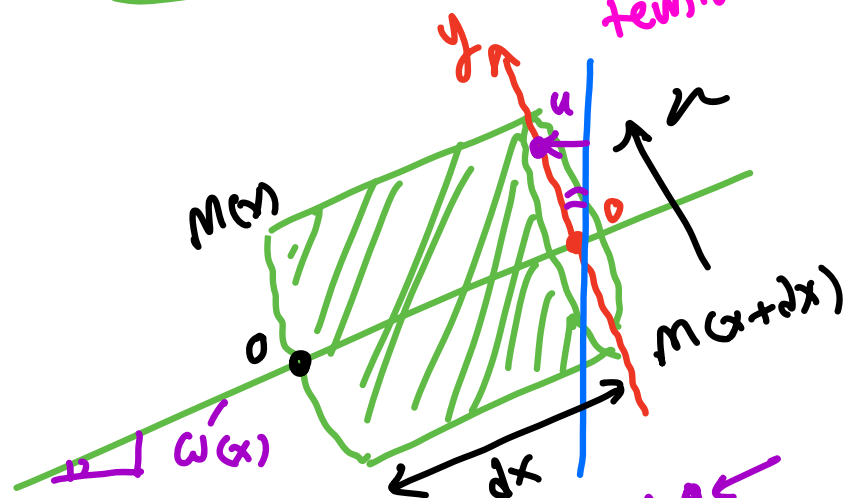
Ex: Euler-Bernoulli Beam Theory



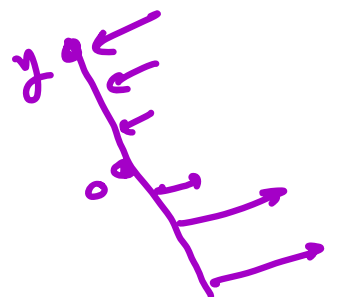
$$u(x, y) = -w'(x)y$$

$$\epsilon(x, y) = \frac{\partial u}{\partial x}$$

$$= -y w''$$



$$\sigma(x, y) = E \epsilon(x, y) = -E y w''$$



$$\int_A \delta = -E\omega'' \int_A y = 0$$

$$\boxed{M} = - \int_A \delta y = E\omega'' \int_A y^2 = \boxed{EI\omega''}$$

Bending moment

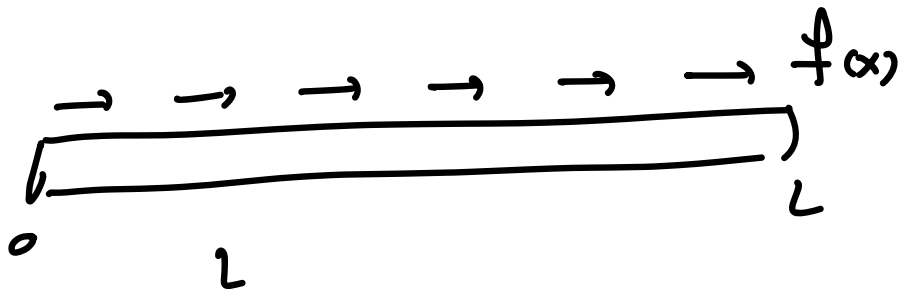
I moment of inertia

$$V = ? \rightarrow M(x+dx) - M(x) + Vdx = 0$$

$= dM$

$$\rightarrow V = - \frac{dM}{dx} \rightarrow \boxed{V = -M'}$$

Recall: For a bar under tension



$$\Pi(u) \equiv \int_0^L \left[\frac{1}{2} EA \epsilon^2 - f u \right] dx \rightarrow \delta \Pi = 0$$

$\epsilon = u'$

$\Pi(u, u') \rightarrow (EA u')' + f = 0$

$$\boxed{\Pi[\omega]} = \int_0^L \int_A \frac{1}{2} E \epsilon^2 - \int_0^L q \omega = \int_0^L \underbrace{\left(\frac{1}{2} E I \omega''^2 - q \omega \right) dx}_{F(\omega, \omega'')} \quad \text{Subst.}$$

$$\delta \Pi(\omega) = \int_0^L \underbrace{(E I \omega'' - q)}_{=0} \delta \omega \, dx + \underbrace{E I \omega'' \delta \omega \Big|_0^L - E I \omega''' \delta \omega \Big|_0^L}_{=0} = 0$$

$$\underline{E-L}: (E I \omega'')' = q$$

Compatible BCs

- ① Clamped : $\omega(0) = 0$, $\omega'(0) = 0$
- ② Simply supported : $\omega(0) = 0$, $M = 0$
 $\omega''(0) = 0$
- ③ Fully free : $M = V = 0$
 $\rightarrow \omega''(0) = 0$, $\omega'''(0) = 0$

5. Higher-order variation, minimums

Def. 2nd variation

$$\delta^2 J \equiv \left. \frac{d^2}{d\epsilon^2} J[u + \epsilon \eta] \right|_{\epsilon=0}$$

Note: Taylor Expand in ϵ

$$J[u + \epsilon \eta] = J[u] + \delta J \epsilon + \delta^2 J \frac{\epsilon^2}{2} + O(\epsilon^3)$$

Note: Can compute $\delta^2 J$ by applying the δ -rule to δJ .

Ex: $J[u] = \int_0^L F(x, u, u') dx$

$$\xrightarrow{\delta} \delta J = \int_0^L \delta F dx$$

$$= \int_0^L (F_u \delta u + F_{u'} \delta u') dx$$

$$\delta \rightarrow \delta^2 J = \int_0^L F_{uu} \delta u^2 + \cancel{F_{uu} \delta u \delta u'} + \cancel{F_{u'u} \delta u' \delta u} + F_{u'u'} \delta u'^2$$

$$2 F_{u'u'} \delta u \delta u'$$

Ex: Consider 1D bar under tension

$$\Pi[u] = \int_0^L \left(\frac{1}{2} EA u'^2 - f u \right) dx$$

$$\delta \rightarrow \boxed{\delta \Pi = \int_0^L EA u' \delta u' - f \delta u} = 0 \quad \delta \delta u' = 0$$

$$\delta \rightarrow \delta^2 \Pi = \int_0^L EA \delta u' \delta u' \geq 0$$

$$\delta \rightarrow \delta^3 \Pi = 0$$

$$\rightarrow \Pi[u + \epsilon \eta] - \Pi[u] + \cancel{\delta \Pi \epsilon} = \delta^2 \Pi \frac{\epsilon^2}{2} > 0$$

if u satisfies $\delta \Pi = 0$

$$\rightarrow \boxed{\Pi[u + \epsilon \eta] \geq \Pi[u]}$$

Satisfies $\delta \Pi = 0$

Remark: Weak form

$$\int_0^L EA u' \delta u' - \int f \delta u = 0$$

trial func. $\delta u'$ test func.

$\delta u'$ $\equiv$ lower regularity demanded

Virtual Work and Energy Thms

1. Two notions of "Work", and Strain Energy

Def. Thermodynamic Work

1st law

$$U = W_{act} + Q$$

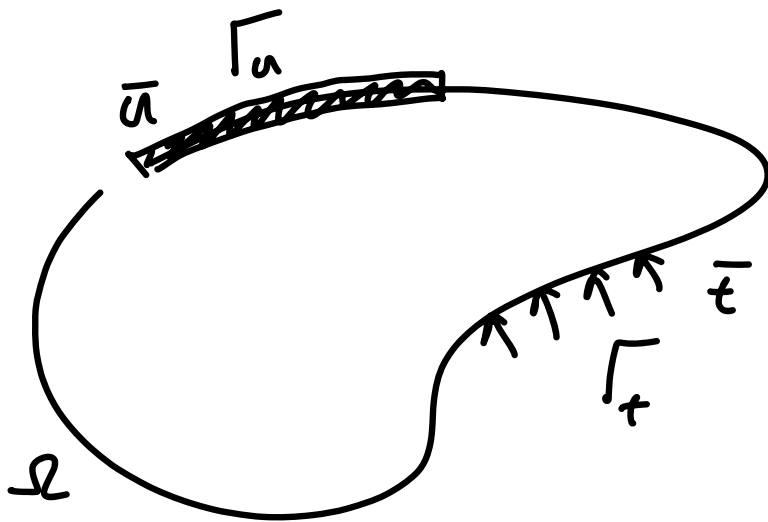
internal Energy work done heat

$$W_{act} = \int_{\text{loading path}} f \cdot du$$

Def. Virtual work (or load functional)

$$W_{\text{ext}}[u] = \int_{\Omega} b \cdot u \, dV + \int_{\Gamma_t} \bar{t} \cdot u \, dA$$

$\forall u$ admissible " u "
($u=0$ on Γ_u)



$$\partial\Omega = \Gamma_u \cup \Gamma_t$$

Def. Strain Energy functional

hyperelastic material, there exists Ψ s.t.

$$b = \frac{\partial \Psi}{\partial \epsilon}, \quad \Psi = \frac{\text{energy stored}}{\text{bulk vol.}}$$

$$U[u] = \int_{\Omega} \Psi(\epsilon) \, dV$$

total Energy stored (elastically)

2. Total potential, Principle of Virtual work

Def.

$$\delta \Pi = 0$$

$$\Pi[u] = U[u] - W_{\text{ext}}[u]$$

Thm. Principle of Virtual Work (PVW)

$$\delta \Pi = 0 \iff \text{Weak form} \iff \text{Strong form}$$

$$\int_{\Omega} \delta : \delta \epsilon - \int_{\Omega} b \cdot \delta u - \int_{\Gamma_t} \bar{t} \cdot \delta u = 0$$

solved in finite elements

$$\begin{aligned} \nabla \cdot b + b &= 0 \\ b n &= \bar{t} \text{ on } \Gamma_t \end{aligned}$$

linear mom. Balance

pf.

$$b = \frac{\partial \psi}{\partial \epsilon} : \delta \epsilon$$

$$\delta \Pi = \int_{\Omega} \delta \psi(\epsilon) - \int_{\Omega} b \cdot \delta u - \int_{\Gamma_t} \bar{t} \cdot \delta u$$

$$\delta \rightarrow \delta \Pi = \int_{\Omega} \underbrace{\delta : \delta \epsilon}_{\substack{\uparrow \\ \text{sym}}} - \int_{\Omega} b \cdot \delta u - \int_{\Gamma_t} \bar{t} \cdot \delta u = 0$$

(Weak $\Rightarrow$ Strong)

$$\delta \epsilon = \delta \nabla_s u = \nabla_s \delta u$$

$\downarrow$
sym.

$$\boxed{\delta : \delta \epsilon = \delta : \nabla \delta u}$$

$$\int \underbrace{\delta : \nabla \delta u}_{//}$$

$$\underbrace{\int_{\Omega} \nabla \cdot (\delta \delta u) - \int_{\Omega} (\nabla \cdot \delta) \delta u}_{//}$$

$$\int_{\Gamma_t} \delta n \cdot \delta u$$

collecting $\rightarrow$

$$- \int_{\Omega} \underbrace{(\nabla \cdot \delta + b)}_0 \delta u + \int_{\Gamma_t} \underbrace{(\delta n - \bar{t})}_0 \delta u = 0$$

□

Thm : Clapyron's Thm

$$U = W_{act} = \frac{1}{2} W_{ext}$$

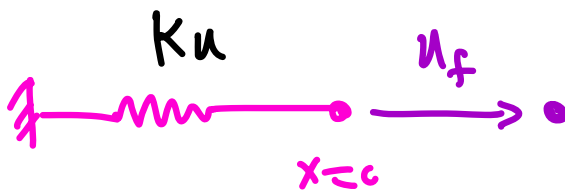
(assuming $Q=0$)

Cauchy

if $u=0$ at $\sqrt{u}$

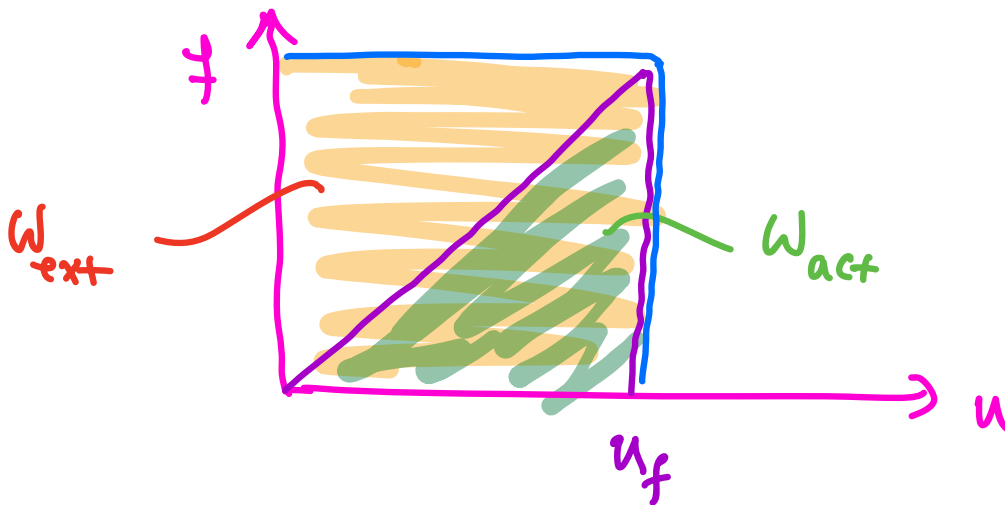
+
Linear Elastic Body

Ex:



$$\left\{ \begin{array}{l} W_{act} = \int_0^{u_f} K u du = \frac{1}{2} K u_f^2 \\ W_{ext} = f \cdot u = K u^2 \end{array} \right.$$

$$\rightarrow W_{act} = \frac{1}{2} W_{ext}$$



p.f. From weak form:

$$\int_{\Omega} b : \delta \epsilon = W_{ext} [\delta u]$$

$\downarrow$
 ϵ

$\downarrow$
 u

solution ϵ - L

admissible

$$\rightarrow \int_{\Omega} \overset{\text{admissible}}{\delta \epsilon} = W_{\text{ext}}$$

$$\underline{\delta = \mathbb{C} : \epsilon}$$

linear

$$\int_{\Omega} \epsilon : \mathbb{C} : \epsilon = 2 \int_{\Omega} \psi = 2U = 2W_{\text{act}} \quad \square$$

1st law

Thm. $\delta \Pi = 0 \Rightarrow u$ st. it minimizes $\Pi[u]$

provided $\mathbb{C}$ is positive definite

$$\delta \epsilon : \frac{\partial \delta}{\partial \epsilon}$$

$$\delta \delta : \delta \epsilon$$

$$(\epsilon : \mathbb{C} : \epsilon > 0, \forall \epsilon)$$

pf.

$$\delta^2 \Pi = \int_{\Omega} \psi - \cancel{\int_{\Omega} \delta b \cdot u} - \cancel{\int_{\Gamma_t} \bar{t} \cdot u}$$

$$\rightarrow \delta^2 \Pi = \int_{\Omega} \delta \epsilon : \underbrace{\frac{\partial \delta}{\partial \epsilon}}_{\mathbb{C}} : \delta \epsilon > 0$$

$$\delta = \mathbb{C} : \epsilon$$

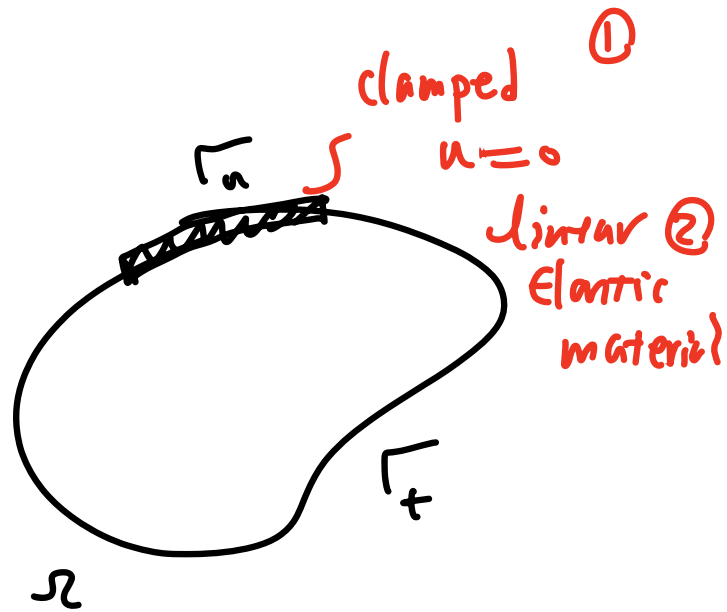
$\rightarrow u$ minimizes $\Pi \quad \square$

3. Reciprocity & Castigliano's thms

Thm. Betti's reciprocity

$$\textcircled{1} (\bar{t}_1, b_1) \rightarrow u_1(x)$$

$$\textcircled{2} (\bar{t}_2, b_2) \rightarrow u_2(x)$$



$$W_{\text{ext}}^1[u_2] = W_{\text{ext}}^2[u_1]$$

$$\rightarrow \int_{\Omega} b_1 \cdot u_2 + \int_{\Gamma_t} \bar{t}_1 \cdot u_2 = \int_{\Omega} b_2 \cdot u_1 + \int_{\Gamma_t} \bar{t}_2 \cdot u_1$$

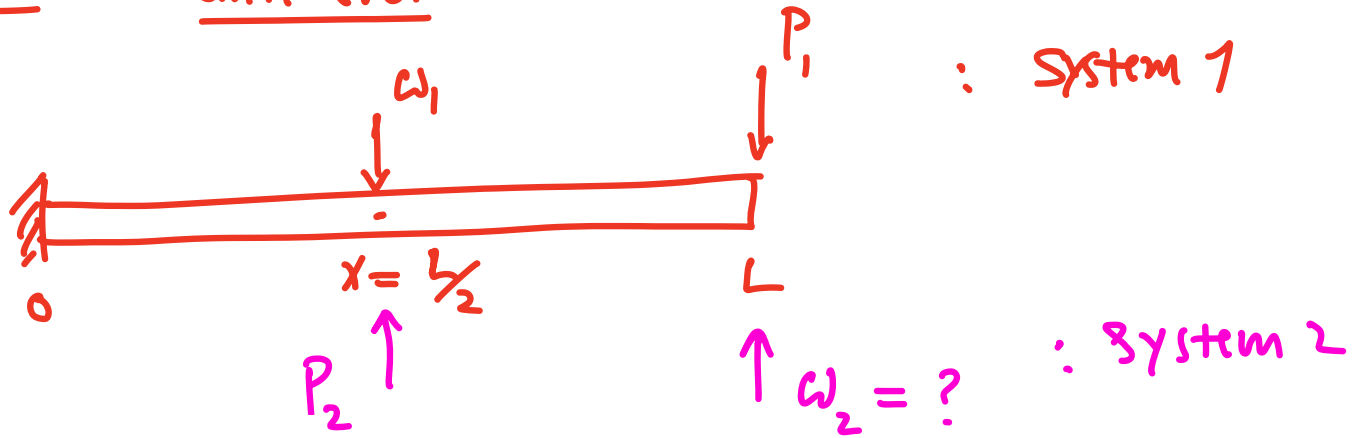
pf. u_1 & u_2 admissible variations

$$\int_{\Omega} b^i : \delta \epsilon = W_{\text{ext}}^i[\delta u] \quad i=1,2$$

$$\int_{\Omega} b^i : \epsilon_2 = \int_{\Omega} \epsilon_1 : c : \epsilon_2 = \int_{\Omega} b^2 : \epsilon_1 \quad \square$$

$C: \epsilon_1$

Ex: Cantilever

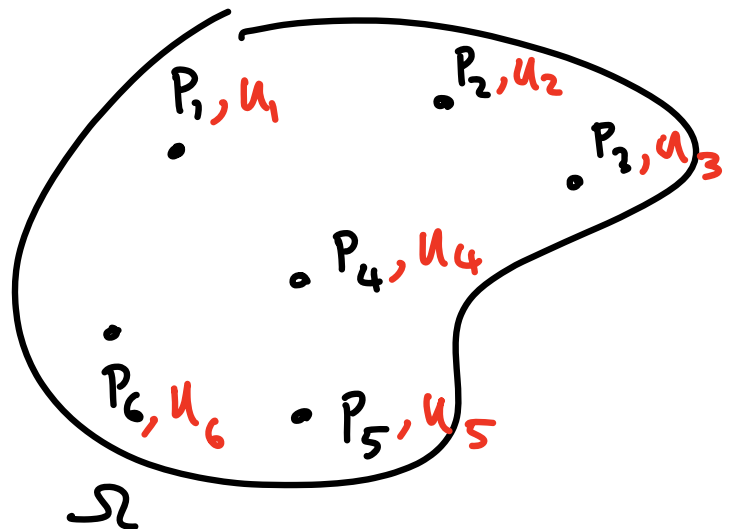


Betti says: $P_1 w_2 = P_2 w_1 \rightarrow w_2 = \frac{P_2 w_1}{P_1}$

Point-load system:

$$U = U(u_1, u_2, \dots, u_n)$$

$$W_{\text{ext}} = \sum_i P_i u_i$$



$$\Pi(u_1, u_2, \dots, u_n) = U - \sum_i P_i u_i$$

$$\nabla \Pi = 0 \rightarrow \frac{\partial \Pi}{\partial u_i} = 0$$

$$\rightarrow \boxed{\frac{\partial U}{\partial u_i} = P_i} \quad \textcircled{\text{I}}$$

What if $\boxed{u_i \doteq \frac{\partial U^*}{\partial P_i}} \quad \textcircled{\text{II}}$

$$U^*(P_1, P_2, \dots, P_n) \equiv \sum_j \underline{P_j} \underline{u_j} - U(u_1, \dots, u_n)$$

$$\rightarrow \boxed{\frac{\partial U^*}{\partial P_i} = u_i} + \cancel{\sum_j P_j \frac{\partial u_j}{\partial P_i}} - \sum_j \cancel{\frac{\partial U}{\partial u_j} \frac{\partial u_j}{\partial P_i}} \quad \begin{matrix} P_j \\ \parallel \\ \frac{\partial U}{\partial u_j} \end{matrix}$$

Def. Linear Structure

$$\vec{P} = K \vec{U} \rightarrow P_i = \sum_j K_{ij} u_j$$

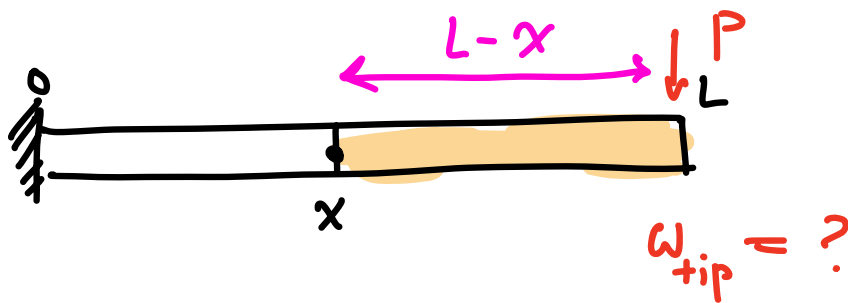
prop. $\boxed{U = \frac{1}{2} \sum_{i,j} u_i K_{ij} u_j = \frac{1}{2} \sum_i P_i u_i}$

$$\boxed{U^* = U}$$

$$U^* = \sum_i P_i \cdot u_i - U = \frac{1}{2} \sum_i P_i \cdot u_i = U$$

$$P_i = \frac{\partial U}{\partial u_i}, \quad u_i = \frac{\partial U^*}{\partial P_i} = \frac{\partial U}{\partial P_i}$$

Ex: Beam / Cantilever tip deflection



$$M(x) = P(L-x)$$

||

$$EI \omega''$$

$$\rightarrow \omega'' = \frac{P}{EI} (L-x)$$

$$\rightarrow \omega(x) = \frac{Px^2(3L-x)}{6EI}$$

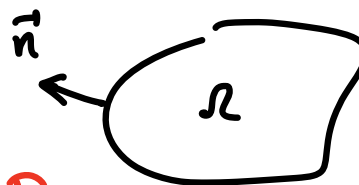
$$U = \int_0^L \frac{1}{2} EI \omega''^2 = \frac{P^2 L^3}{6EI}$$

$$\boxed{u_{tip} = \frac{\partial U^*}{\partial P} = \frac{\partial U}{\partial P} = \frac{PL^3}{3EI}}$$

Thermodynamics & Dissipation Potentials

1. Two laws of thermo (in power), dissipation inequality

Axiom. 1st law



$$\frac{d}{dt} \int_{\Omega} e \, dv = \cancel{P_{act}} - \int_{\partial\Omega} q \cdot n \, dA + \int_{\Omega} r \, dv$$

$\int_{\Omega} e \, dv$: internal energy bulk vol.
 $\int_{\partial\Omega} q \cdot n \, dA$: heat flux
 $\int_{\Omega} r \, dv$: heat rate vol.

$$P_{act} = P_{ext} \quad (W_{act} \neq W_{ext})$$

$$W_{act} = \int_0^t \int_{\Omega} f \cdot \dot{u} \, dt \rightarrow \dot{W}_{act} = \int_{\Omega} f \cdot \dot{u} \equiv P_{ext}$$

$$P_{\text{ext}} = \int_{\Omega} b \cdot \dot{u} + \int_{\Gamma_t} \bar{t} \cdot \dot{u} \stackrel{\text{weak form}}{=} \int_{\Omega} b : \dot{\epsilon} = P_{\text{int}}$$

Prop. local form of 1st-law

$$\dot{\epsilon} = b : \dot{\epsilon} - \nabla \cdot q + r$$

pf.

$$\int_{\Omega} \underbrace{(\dot{\epsilon} - b : \dot{\epsilon} + \nabla \cdot q - r)}_0 dV = 0 \quad \forall \Omega$$

□

Axiom : 2nd law of thermo

$$\dot{s} = - \nabla \cdot \left(\frac{q}{T} \right) + \frac{r}{T} + \underbrace{\dot{s}_G}_{\geq 0}$$

$$\rightarrow \boxed{\dot{s} \geq - \nabla \cdot \left(\frac{q}{T} \right) + \frac{r}{T}}$$

Def. Helmholtz free energy

$$\underbrace{\Psi}_{\text{helmholtz}} \equiv e - TS \quad \text{legendre transf.}$$

Thm. Clausius-Duhem inequality

$$\boxed{Q \equiv \dot{b} : \dot{e} - \dot{\Psi} - s \dot{T} - \frac{q \cdot \nabla T}{T} \geq 0}$$

pf.

$$T \left(\begin{aligned} \dot{s} &\geq - \nabla \cdot \left(\frac{q}{T} \right) \\ &= - \frac{\nabla \cdot q}{T} + \frac{\nabla T \cdot q}{T^2} \end{aligned} \right)$$

lets ignore
"r"
for simplicity

$$\rightarrow T \dot{s} \geq - \underbrace{\nabla \cdot q}_{\dot{b} : \dot{e} - \dot{e}} + \frac{q \cdot \nabla T}{T}$$

$$\dot{\Psi} - T\dot{s} - s\dot{T}$$

$$\rightarrow \cancel{T\dot{s}} \geq -\dot{\epsilon} - \cancel{\dot{\Psi}} + \cancel{T\dot{s}} + \underline{s\dot{T}} + \frac{g \cdot \nabla T}{T}$$

$$\rightarrow \dot{\epsilon} - \dot{\Psi} - s\dot{T} - \frac{g \cdot \nabla T}{T} \geq 0$$

□

Corollary: Constant & uniform ($T = \text{cte}$)

$$\boxed{\mathcal{D} = \dot{\epsilon} - \dot{\Psi} \geq 0}$$

Remark: For hyperelastic material $\Psi = \Psi(\epsilon)$
 $\underline{\dot{\epsilon} = \dot{\epsilon}(\epsilon)}$

$$\dot{\Psi} = \frac{\partial \Psi}{\partial \epsilon} : \dot{\epsilon}$$

$(\dot{T}, \dot{\epsilon})$ are
externally
controllable
variables

$$\rightarrow \mathcal{D} = \left(\underline{\dot{\epsilon}(\epsilon)} - \frac{\partial \Psi}{\partial \epsilon} \right) : \dot{\epsilon} \geq 0$$

$\neq \dot{\epsilon}(\epsilon)$ arbitrary

$$\rightarrow \boxed{\sigma = \frac{\partial \psi}{\partial \epsilon}} \rightarrow \boxed{D = 0}$$

2. Coleman-Noll procedure

Thermo-elastic medium

$$\left\{ \begin{array}{l} \psi = \psi(\epsilon, T) \\ \sigma = \sigma(\epsilon, T) \\ s = s(\epsilon, T) \\ q = q(\epsilon, T, \nabla T) \\ \dot{\epsilon}, \dot{T} = \text{externally controllable} \end{array} \right.$$

$$D = \sigma : \dot{\epsilon} - \dot{\psi} - s \dot{T} - \frac{q \cdot \nabla T}{T} \geq 0$$

$$\dot{\psi} = \frac{\partial \psi}{\partial \epsilon} : \dot{\epsilon} + \frac{\partial \psi}{\partial T} \dot{T}$$

$$\rightarrow D = \underbrace{\left(\sigma - \frac{\partial \psi}{\partial \epsilon} \right)}_{\neq f(\dot{\epsilon})} : \overset{\text{arbitrary}}{\dot{\epsilon}} - \underbrace{\left(s + \frac{\partial \psi}{\partial T} \right)}_{\neq f(\dot{T})} \dot{T} - \frac{q \cdot \nabla T}{T} \geq 0$$

↑ arbitrary

$$\rightarrow \boxed{\begin{aligned} \mathcal{D} &= - \frac{q \cdot \nabla T}{T} \geq 0 \\ \sigma &= \frac{\partial \psi}{\partial \epsilon}, \quad s = - \frac{\partial \psi}{\partial T} \end{aligned}}$$

Corollary: Maxwell's relation

$$\frac{\partial^2 \psi}{\partial \epsilon \partial T} = \frac{\partial^2 \psi}{\partial T \partial \epsilon} \rightarrow \boxed{\frac{\partial \sigma}{\partial T} = - \frac{\partial s}{\partial \epsilon}}$$

Strain-rate dependent processes ($T = \text{const}$)

$$\psi = \psi(\epsilon, \dot{\epsilon})$$

$$\sigma = \sigma(\epsilon, \dot{\epsilon})$$

$$\text{dissipation} \quad \mathcal{D} = \sigma : \dot{\epsilon} - \dot{\psi} \geq 0$$

$$\dot{\psi} = \frac{\partial \psi}{\partial \epsilon} : \dot{\epsilon} + \frac{\partial \psi}{\partial \dot{\epsilon}} : \ddot{\epsilon}$$

$$\rightarrow Q = \underbrace{\left(\underbrace{\sigma - \frac{\partial \psi}{\partial \epsilon}}_{= \tau(\dot{\epsilon}, \epsilon)} \right) : \dot{\epsilon}}_{\substack{\text{viscom stress} \\ \text{arbitrary Controllable}}} - \underbrace{\frac{\partial \psi}{\partial \dot{\epsilon}} : \ddot{\epsilon}}_{\substack{\text{arbitrary} \\ \neq \tau(\ddot{\epsilon})}} \geq 0$$

$$\rightarrow \boxed{\begin{aligned} Q &= \sigma_v : \dot{\epsilon} \geq 0 \\ \frac{\partial \psi}{\partial \dot{\epsilon}} &= 0, \quad \sigma = \frac{\partial \psi}{\partial \epsilon} + \sigma_v \end{aligned}}$$

3. Internal Variables, dissipation Potential (T=du)

$$\psi = \psi(\epsilon, \xi) \quad \text{internal variable } (\xi)$$

$$\sigma = \sigma(\epsilon, \xi)$$

$\xi, \dot{\xi}, \dots =$ cannot be externally controlled

$\xi =$ captures internal changes

(yield, fracture, dislocation / grain rearrangements)

prop. $\Psi = \Psi(\epsilon, \dot{\epsilon}, \xi)$
 $\delta = \delta(\epsilon, \dot{\epsilon}, \xi)$

$$\mathcal{D} = \delta : \dot{\epsilon} - \dot{\Psi} \geq 0$$

$$\dot{\Psi} = \frac{\partial \Psi}{\partial \epsilon} : \dot{\epsilon} + \frac{\partial \Psi}{\partial \dot{\epsilon}} : \ddot{\epsilon} + \frac{\partial \Psi}{\partial \xi} \dot{\xi}$$

$$\rightarrow \mathcal{D} = \underbrace{\left(\delta - \frac{\partial \Psi}{\partial \epsilon} \right)}_{= \gamma(\dot{\epsilon})} : \dot{\epsilon} - \underbrace{\frac{\partial \Psi}{\partial \dot{\epsilon}}}_{\delta} : \ddot{\epsilon} - \underbrace{\frac{\partial \Psi}{\partial \xi}}_{X} \dot{\xi} \geq 0$$

"flux"
"conjugate force"

$$\rightarrow \left\{ \begin{array}{l} \mathcal{D} = \delta_r : \dot{\epsilon} + X \dot{\xi} \geq 0 \\ \frac{\partial \Psi}{\partial \dot{\epsilon}} = 0, \quad \delta = \frac{\partial \Psi}{\partial \epsilon} + \delta_r \end{array} \right.$$

Def. often terms like F.I appear in $\mathcal{D}$.

$$\begin{cases} F = \text{Conjugate force} \\ J = \text{flux} \end{cases}$$

$$\begin{matrix} (\zeta_v, \epsilon) & (X, \dot{\xi}) \\ \textcolor{red}{F}'' & \textcolor{red}{J}'' \end{matrix}$$

A dissipation potential is a function defined / corresponding to the pair (F, J)

$\Phi(J) \geq 0$, $\Phi(0) = 0$, Convex, $F = \frac{\partial \Phi}{\partial J}$

dissipation potential

Thm. For a dissip. potential $\Phi(J)$

$$\textcolor{blue}{F \cdot J} = \frac{\partial \Phi}{\partial J} \cdot J \geq \textcolor{blue}{\Phi(J)} \geq 0$$

pf.

$$\Phi(J) + \frac{\partial \Phi}{\partial J} (y - J) \leq \Phi(y)$$

Pick $y = 6$

END of Day 2
